

# Lecture 04 - Finite Differences for Modelling Heat Conduction

June 17, 2025

# Outline

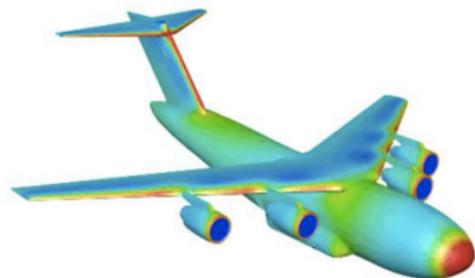
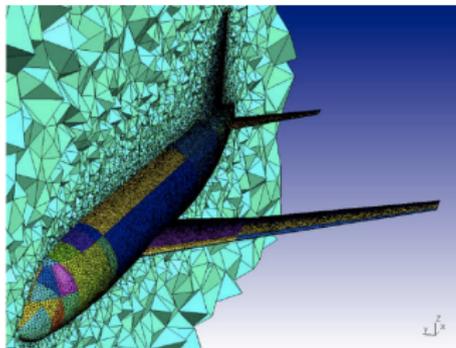
- 1 Finite Differences for Modelling Heat Conduction

# Finite Differences for Modelling Heat Conduction

This lecture covers an application of solving linear systems. Partial differential equations (PDEs) involve multivariable functions and (partial) derivatives. They describe numerous phenomena:

- Electromagnetism,
- Fluid flow,
- Sound propagation,
- Financial problems,
- Solid mechanics  
(engineering),
- Quantum mechanics,
- ...

# Finite Differences for Modelling Heat Conduction



The numerical solution of PDEs are a common source of sparse linear systems (e.g., finite difference/finite volume/finite element methods). This lecture introduces finite differences for a PDE describing heat conduction.

# Finite Differences for Modelling Heat Conduction

## Setup:

- 1 Suppose that we want to approximate the (unknown) **temperature function**,  $T(x, y, z)$  (where  $x, y, z$  are spatial coordinates) in some 3-D solid object, **at equilibrium** (i.e.  $T$  does not vary with respect to time).
- 2 Suppose further that we have a given (known) **heat source function**,  $f(x, y, z)$ .

# Finite Differences for Modelling Heat Conduction

Then the heat distribution may be modelled using the **Poisson equation**:

$$\begin{aligned} f + \left( \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} \right) & \stackrel{\text{at equilibrium}}{=} 0 \\ \Leftrightarrow - \left( \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} \right) & = f \\ \Leftrightarrow -\Delta T & = f. \end{aligned}$$

The differential operator

$$\Delta = \left( \frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} + \frac{\partial^2}{\partial z^2} \right)$$

is called the **Laplacian operator**.

## Finite Differences for Modelling Heat Conduction

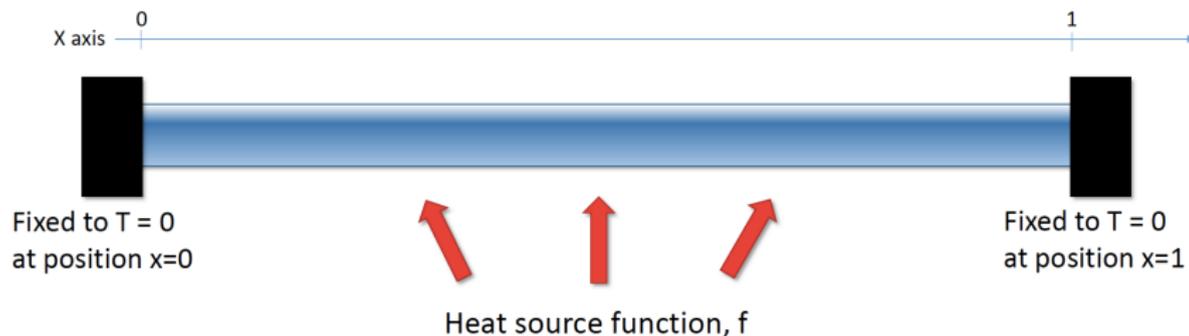
**1D Example** Boundary conditions are also necessary to fully define the problem. Consider a 1D example where

$$-\frac{\partial^2 T}{\partial x^2} = f \quad \text{on } (0, 1), \quad (1)$$

$$T(0) = 0, \quad (2)$$

$$T(1) = 0. \quad (3)$$

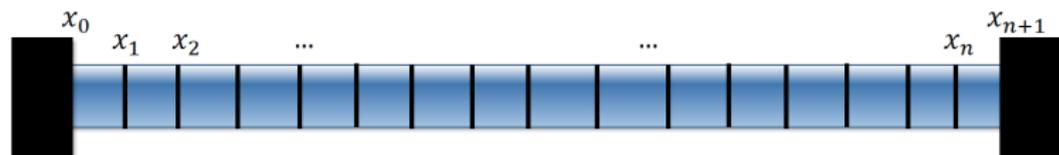
Lines (2) and (3) are the boundary conditions. In this case the temperature  $T$  is zero at both  $x = 0$  and  $x = 1$ . The figure below shows the domain pictorially.



## Finite Differences for Modelling Heat Conduction

We want to find an approximate numerical solution, given the source  $f$  and the boundary temperatures. Our approach here is to first discretize (subdivide) the material into finite subintervals. Then approximate the spatial derivatives with finite differences.

**Discretizing** the domain is done by chopping the length of the 1D bar into chunks. That is, define discrete points on the bar  $0 = x_0 < x_1 < x_2 < \dots < x_n < x_{n+1} = 1$ , which are referred to as **gridpoints**  $x_i$ .



We let  $T_i$  denote the numerical approximation of the exact solution  $T(x_i)$  for  $i = 0, \dots, n$ .

# Finite Differences for Modelling Heat Conduction

Here we assume evenly spaced intervals. Define the grid spacing  $h$  as

$$\begin{aligned} h = x_i - x_{i-1} &= \frac{1}{n+1}, \\ &= \frac{\text{domain length}}{\# \text{ of intervals}}. \end{aligned}$$

Due to the boundary conditions we know  $T_0 = 0$  and  $T_{n+1} = 0$ . Therefore, the unknowns we must solve for are the  $n$  temperature values  $T_1, T_2, \dots, T_n$  (i.e., at non-boundary gridpoints). These gridpoints are referred to as the **active** gridpoints.

# Finite Differences for Modelling Heat Conduction

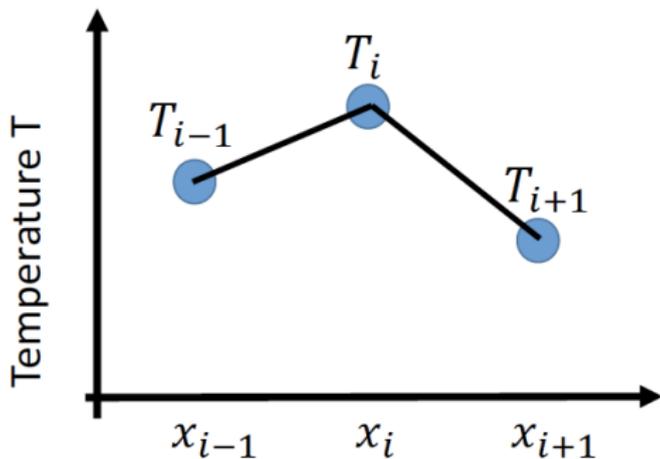
Now that the discrete domain is defined we discretize the partial derivatives. Recall, **finite differences** are one approach to obtain discrete approximations of derivatives. For example,

$$\frac{\partial T}{\partial x}(x_i) \approx \frac{T(x_i) - T(x_{i-1})}{x_i - x_{i-1}} \approx \frac{T_i - T_{i-1}}{h}. \quad (4)$$

## Finite Differences for Modelling Heat Conduction

We will use the **centered finite difference** approximation of  $\frac{\partial^2 T}{\partial x^2}$ , specifically

$$\begin{aligned} & \frac{\partial^2 T}{\partial x^2}(x_i) \\ & \approx \frac{\frac{T_{i+1} - T_i}{h} - \frac{T_i - T_{i-1}}{h}}{h} \\ & = \frac{T_{i+1} - 2T_i + T_{i-1}}{h^2} \end{aligned} \tag{5}$$



## Finite Differences for Modelling Heat Conduction

It can be seen from (5), and the above figure, that  $T_i$  depends on its neighbours  $T_{i+1}$  and  $T_{i-1}$ . The resulting relationships between gridpoints determines  $T_i$ , for  $i = 1, 2, \dots, n$ . Each gridpoint  $i = 1, 2, \dots, n$  gives one equation relating its value to its two neighbours:

$$-\left(\frac{T_{i+1} - 2T_i + T_{i-1}}{h^2}\right) = f_i, \quad \text{for } i = 1, \dots, n. \quad (6)$$

Equation (6) is our **discrete** equation approximating the **continuous** equation  $-\frac{\partial^2 T}{\partial x^2} = f$ .

The general form of the matrix equation from (6) is

$$\frac{1}{h^2} \begin{bmatrix} 2 & -1 & & & & \\ -1 & 2 & -1 & & & \\ & \ddots & \ddots & \ddots & & \\ & & -1 & 2 & -1 & \\ & & & -1 & 2 \end{bmatrix} \begin{bmatrix} T_1 \\ T_2 \\ \vdots \\ T_{n-1} \\ T_n \end{bmatrix} = \begin{bmatrix} f_1 \\ f_2 \\ \vdots \\ f_{n-1} \\ f_n \end{bmatrix}.$$

# Finite Differences for Modelling Heat Conduction

## How The Boundary Conditions Determine The First And Last Rows

$$\begin{aligned} f_1 &= - \left( \frac{T_2 - 2T_1 + \underbrace{T_0}_{=0}}{h^2} \right) \\ &= \frac{2T_1 - T_2}{h^2} \\ f_n &= - \left( \frac{\underbrace{T_{n+1}}_{=0} - 2T_n + T_{n-1}}{h^2} \right) \\ &= \frac{-T_{n-1} + 2T_n}{h^2} \end{aligned}$$

What can we say about the matrix structure? It is a banded matrix, but more specifically symmetric and tridiagonal.

# Finite Differences for Modelling Heat Conduction

**Heat Conduction in 2D Plate** Consider the 2D domain of a square plate with zero temperature boundaries. We want to determine the heat distribution  $T(x, y)$  on the interior given a heat source function  $f(x, y)$ . Figure 1 shows an example of the 2D plate and a heat distribution for an example  $f$ .

# Finite Differences for Modelling Heat Conduction

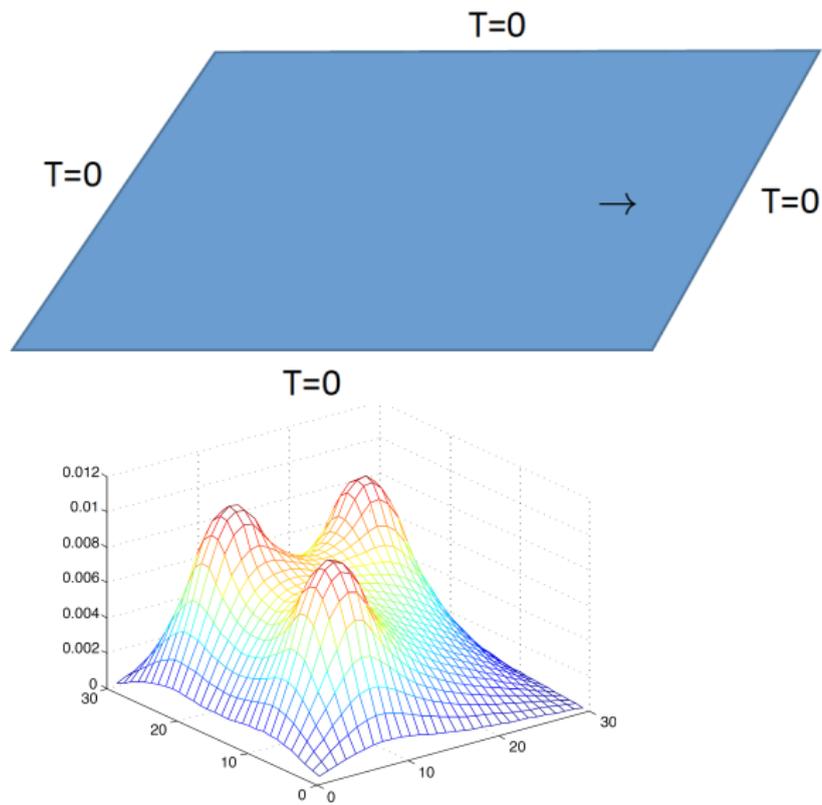


Figure: Two-dimensional plate domain (left) and heat distribution (right).

## Finite Differences for Modelling Heat Conduction

Gridpoints are now indexed by  $i, j$  so that  $(x_i, y_j)$  defines a discrete location on the 2D plate. The inset shows an example grid. The approximate temperature at  $(x_i, y_j)$  is denoted  $T_{i,j}$  such that  $T_{i,j} \approx T(x_i, y_j)$ . We now need to approximate the 2D continuous Poisson equation

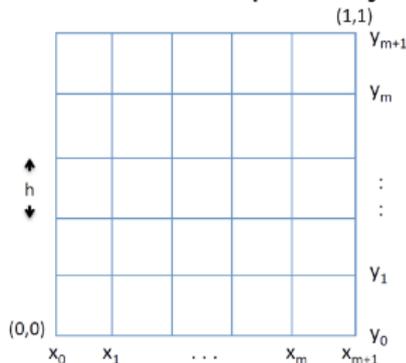
$$-\left(\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2}\right) = f,$$

at each gridpoint.

## Finite Differences for Modelling Heat Conduction

The discrete Poisson equation in 2D is obtained by approximating the 2<sup>nd</sup> derivative in each axis separately, then summing them

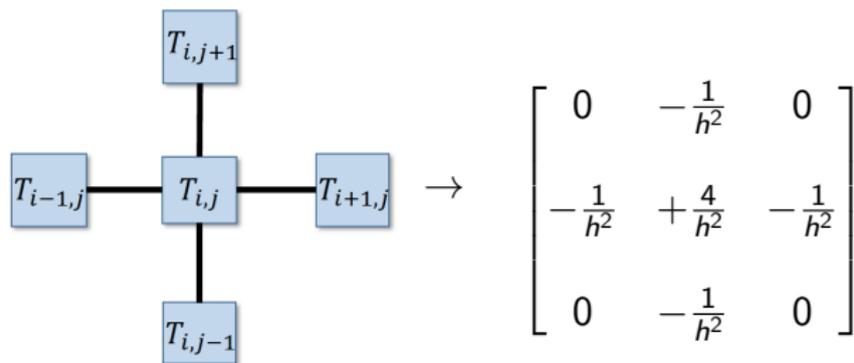
together. That is,



$$\begin{aligned}
 & - \left( \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} \right) = f \\
 - \left( \frac{T_{i+1,j} - 2T_{i,j} + T_{i-1,j}}{h^2} + \frac{T_{i,j+1} - 2T_{i,j} + T_{i,j-1}}{h^2} \right) & = f_{i,j} \\
 \frac{4T_{i,j} - T_{i-1,j} - T_{i+1,j} - T_{i,j-1} - T_{i,j+1}}{h^2} & = f_{i,j} \quad (7)
 \end{aligned}$$

## Finite Differences for Modelling Heat Conduction

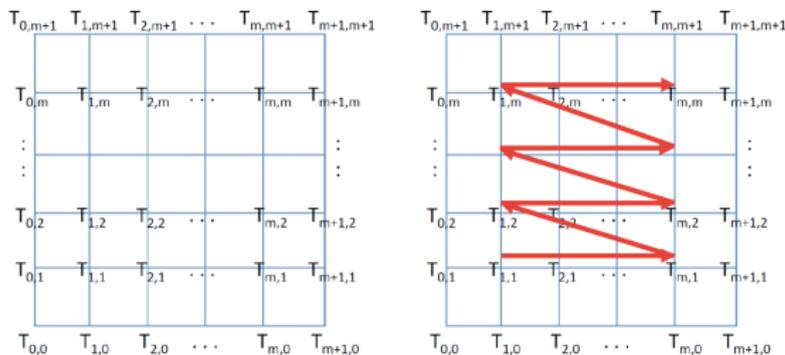
The finite difference **stencil** is a convenient visual notation for (7) centered at each gridpoint (see Figure 2). The nonzeros in the stencil will be the nonzeros in a row of the matrix.



**Figure:** The finite difference stencil for the left hand side of (7), i.e., the **negative** of the 2D discrete Laplacian.

## Finite Differences for Modelling Heat Conduction

To put (7) into matrix form we need to “flatten” the indices from 2D  $(i, j)$  to 1D  $(k)$ . The 2D computational domain is indexed from 0 to  $m + 1$  in each dimension (see Figure 3 left). Since the boundary values are known to be 0, we only need to solve for unknowns  $T_{i,j}$  for all  $i, j \in [1, m]$  (a total of  $m^2$  unknowns). How do we index the unknowns into a 1D array? A natural rowwise ordering numbers gridpoints along the  $x$ -axis first, then along the  $y$ -axis (see Figure 3 right).



**Figure:** Two-dimensional indexing (left) for a discrete plate and a possible 1D ordering/flattening (right).

# Finite Differences for Modelling Heat Conduction

Specifically,

$$T_{1,1} \rightarrow T_1,$$

$$T_{2,1} \rightarrow T_2,$$

$$\vdots$$

$$T_{m,1} \rightarrow T_m,$$

$$T_{1,2} \rightarrow T_{m+1},$$

$$T_{2,2} \rightarrow T_{m+2},$$

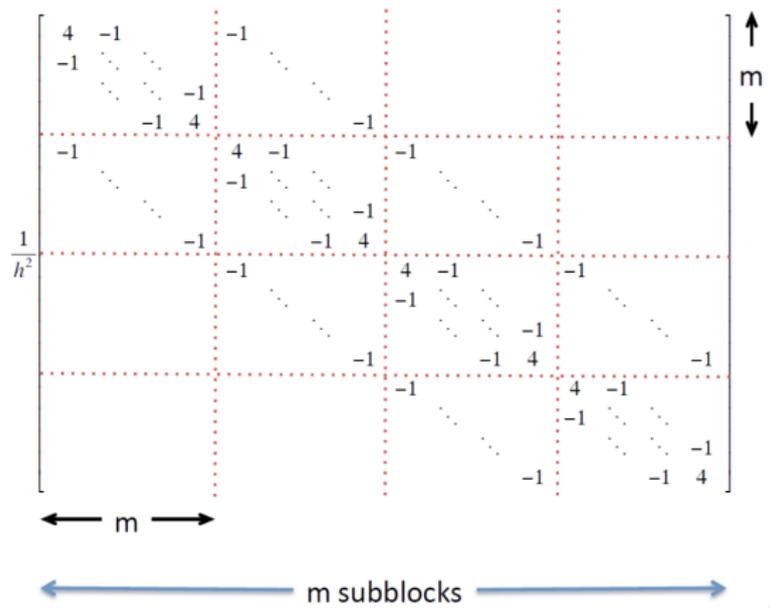
$$\vdots$$

$$T_{m,m} \rightarrow T_{m^2}.$$

That is, we convert from 2D indices  $(i, j)$  to 1D indices  $k$  using  $k = i + (j - 1) \times m$ .

## Finite Differences for Modelling Heat Conduction

The general form of the Laplacian matrix in 2D with this natural rowwise ordering is given in Figure 4.



**Figure:** General matrix structure for discrete Laplacian with natural rowwise ordering.

# Finite Differences for Modelling Heat Conduction

Notice that there are 5 bands:

- 1 diagonal band,
- 2 bands immediately above/below the diagonal,
- 2 bands separated horizontally by  $m$  entries.

# Finite Differences for Modelling Heat Conduction

## Explanation:

- Let  $(i, j)$  be arbitrary,  $1 \leq i \leq m, 1 \leq j \leq m$ .
- Consider the equation

$$\frac{1}{h^2} (4T_{i,j} - T_{i-1,j} - T_{i+1,j} - T_{i,j-1} - T_{i,j+1}) = f_{i,j}.$$

- The 4 coefficient appears on the diagonal, since  $i, j$  on the LHS agrees with  $i, j$  on the RHS.
- The  $-T_{i-1,j}$  term contributes
  - nothing, if  $i = 1$ , by boundary conditions, and
  - $-1$  immediately to the left of the diagonal, otherwise.
- The  $-T_{i+1,j}$  term contributes
  - nothing, if  $i = m$ , by boundary conditions, and
  - $-1$  immediately to the right of the diagonal, otherwise.
- The  $-T_{i,j-1}$  term contributes
  - nothing, if  $j = 1$ , by boundary conditions, and
  - $-1$ ,  $m$  positions to the left of the diagonal, otherwise.
- The  $-T_{i,j+1}$  term contributes
  - nothing, if  $j = m$ , by boundary conditions, and
  - $-1$ ,  $m$  positions to the right of the diagonal, otherwise.

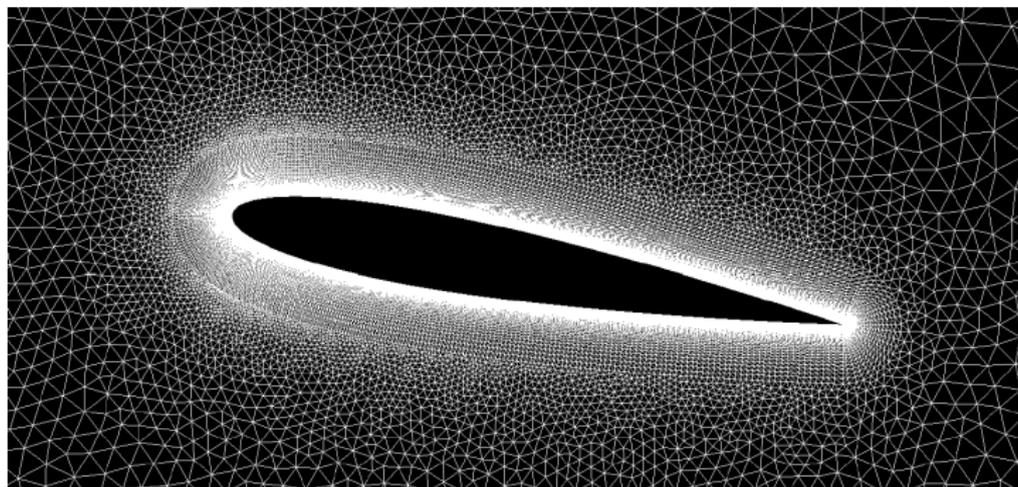
# Finite Differences for Modelling Heat Conduction

## Remarks:

- ① The 2-D setup quickly becomes more complicated than the 1-D setup.
- ② Adding more dimensions would further increase the complexity.
- ③ If we relax our assumption about not allowing changes in temperature over time, then we would need to add a time dimension. E.g. adding a time dimension to the 1-D setup would make it 2-D to start. We would have to be careful to clearly define our boundary conditions in this case.

## Finite Differences for Modelling Heat Conduction

Other types of PDE problems, discretizations, and geometries give rise to different matrix structures and properties. For example, a triangular mesh can be used with a finite volume discretization to study the flow around an airfoil (see Figure 5).



**Figure:** Example discretization using triangles for an airfoil.

# Finite Differences for Modelling Heat Conduction

This lecture only considered modelling heat in an equilibrium using the Poisson equation. The time-dependent heat equation considers non-equilibrium situations, i.e., how temperature evolves over time. The finite difference equations are similar and lead to another linear system to solve.